

# Aspire Energy Resources Ltd.

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<b>Date:</b>	09/07/2012	Rev: <span style="border: 1px solid black; padding: 2px;">0</span>
<b>Client:</b>	Aspire Energy Resources Limited	
<b>Project Number:</b>	4188	
<b>Drawing Number:</b>	CA-4665-03	
<b>A Number:</b>	228554	<b>Approved:</b> _____
<b>CRN:</b>	F-7938.2	
<b>Serial Number:</b>	L4-583	<b>Date:</b> _____

Design in accordance with ASME Section VIII, Division 1:

Edition:	1983
Addenda:	1985

The higher stress values allowed by the 1999 Code Addenda have NOT been applied to this design.

## Inside Diameter Calculations

### Design Data

Design Pressure	P	50	PSIG	
Minimum Design Temperature		-20	F	
Maximum Design Temperature		199	F	Stress Values Are Affected by Temps Over 100 F!
Flange MAWP	Pf	285	PSIG	Flange MAWP Is Affected by Temps Over 100 F!
Inside Diameter	D	48	in	
Inside Radius	R	24	in	
Corrosion Allowance	c	0.125	in	
Shell Material Stress Value	S	17,500	PSI	SA-106-B, Smls - 15,000    SA-516-70 - 17,500
Head Material Stress Value	S	17,500	PSI	SA-234-WPB - 15,000    SA-516-70 - 17,500
Joint Efficiency ( Type 1 Joint )	E	0.85		Full / Partial Radiography: 1.0 or Spot Radiography: Pipe - 1.0, Plate - 0.85 or No Radiography: Pipe - 0.85, Plate - 0.70

### Shell Minimum Thickness

**Shell tr = PR / ( SE - 0.6P )**      = ( 50 \* ( 24 + 0.125 ) ) / ( ( 17500 \* 0.85 ) - ( 0.6 \* 50 ) )

Minimum shell thickness      tr =      0.0813 in

tr + c =      **0.2063** in

Actual shell thickness      UT 0.3060 in

MAWP based on actual thk      **111.10** PSIG

### Head Minimum Thickness

**Head tr = PD / ( 2SE - 0.2P )**      = ( 50 \* ( 48 + 0.125 + 0.125 ) ) / ( ( 2 \* 17500 \* 0.85 ) - ( 0.2 \* 50 ) )

Minimum head thickness      tr =      0.0811 in

tr + c =      **0.2061** in

Actual head thickness      UT 0.4490 in

MAWP based on actual thk      **199.50** PSIG

**MAWP of Vessel:                      111.10 PSIG                      Design Limited by the Shell**